

# Lecture 21

## Optical MEMS (3)

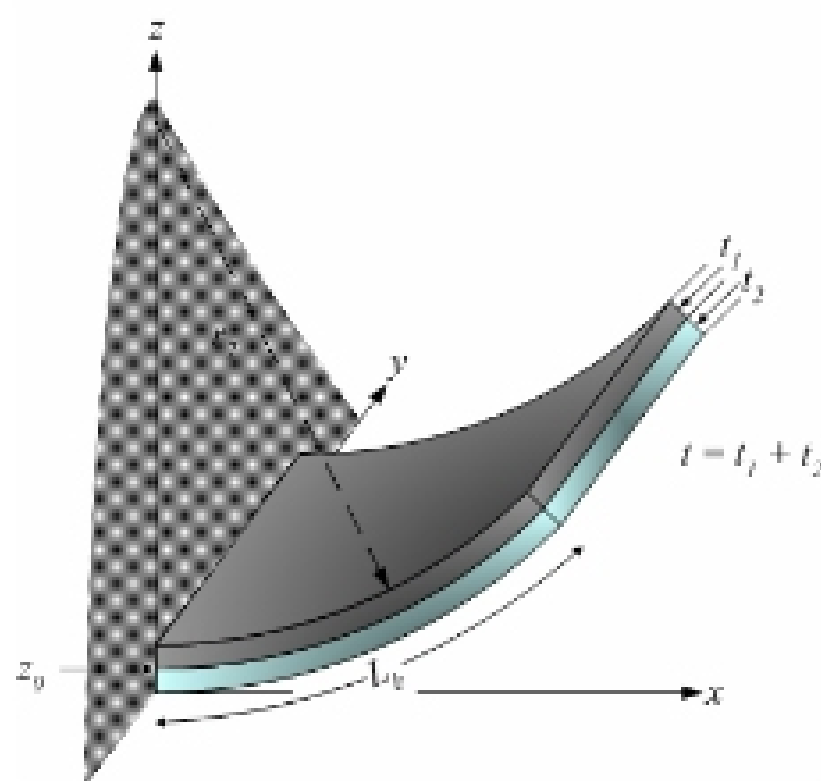
### ■ Agenda:

#### ➤ Micromirrors

##### – Thermal Actuation

- Bimorph
- Hot-arm
- Lateral

## Bimorph Actuator Design



Strain due to temperature change:

$$\begin{cases} \varepsilon_{T1} = \alpha_1 \Delta T \\ \varepsilon_{T2} = \alpha_2 \Delta T \end{cases}$$

Corresponding stress:

$$\begin{cases} \sigma_{T1} = E_1 \varepsilon_{T1} = E_1 \alpha_1 \Delta T \\ \sigma_{T2} = E_2 \varepsilon_{T2} = E_2 \alpha_2 \Delta T \end{cases}$$

$\Delta T$ : temperature change.

$\alpha_1$ : temperature coefficient of expansion of first layer

$\alpha_2$ : temperature coefficient of expansion of second layer

$E_1$ : Young's modulus of first layer

$E_2$ : Young's modulus of second layer

- The deformation of the beam due to  $\Delta T$  causes bending strain inside the beam and the strain varies along the  $z$ -axis.
- Assume the mean strain is  $\varepsilon_0$  and the plane of the mean strain is located at  $z_0$ .
- The **bending strain** can be written as

$$\varepsilon_b(z) = \varepsilon_0 + \frac{z_0 - z}{r} = \left( \varepsilon_0 + \frac{z_0}{r} \right) - \frac{z}{r}$$

where  $r$  is the radius of curvature, and  $z$  is the distance to the bottom of the beam. The **equivalent bending stress** is

$$\sigma_b(z) = E(z) \cdot \varepsilon_b(z)$$

$$E(z) = \begin{cases} E_1 & 0 \leq z < t_1 \\ E_2 & t_1 \leq z \leq t_2 \end{cases}$$

At **equilibrium**, both the total force and the total bending moment should be equal to zero

$$\begin{cases} \sum F = \int_0^t \int_0^w \sigma(z) dy dz = 0 \\ \sum M = \int_0^t \int_0^w \sigma(z) z dy dz = 0 \end{cases}$$

where

$$\sigma(z) = \sigma_T(z) - \sigma_b(z)$$

$$\sigma_T(z) = \begin{cases} \sigma_{T1} & 0 \leq z < t_1 \\ \sigma_{T2} & t_1 \leq z \leq t_1 + t_2 \end{cases}$$

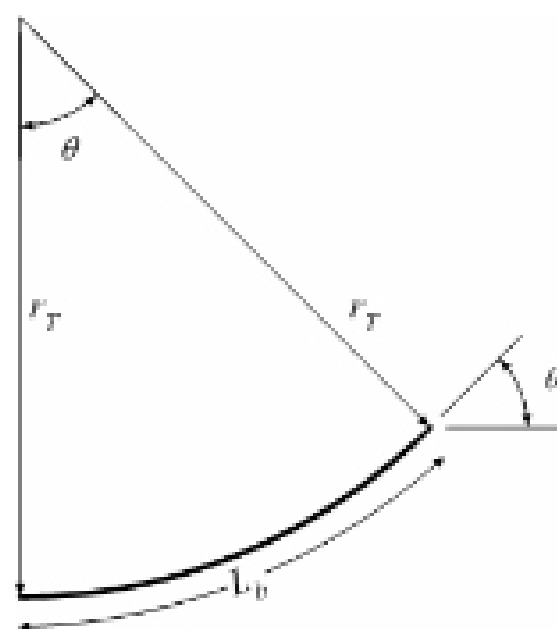
Assuming the temperature and stress are uniform along the y axis, the **radius of curvature** due to temperature change,  $r_T$ , can be obtained by solving the force and moment equations:

$$\frac{1}{r_T} = \frac{6t(\alpha_2 - \alpha_1)\Delta T}{4t_1^2 + 4t_2^2 + 6t_1t_2 + \frac{E_1t_1^3}{E_2t_2} + \frac{E_2t_2^3}{E_1t_1}}$$

$$\frac{1}{r_T} = \beta_r \Delta\alpha_T \Delta T$$

$\Delta\alpha_T = \alpha_2 - \alpha_1$  and  $\beta_r$  is **curvature coefficient**

$$\beta_r \triangleq \frac{6t}{4t_1^2 + 4t_2^2 + 6t_1t_2 + \frac{E_1t_1^3}{E_2t_2} + \frac{E_2t_2^3}{E_1t_1}}$$



Tangential angle at the beam end is given by

$$\theta = \frac{L_b}{r_T}$$

Substituting the expression for  $r_T$  yields

$$\theta(\Delta T) = \beta_r L_b \Delta\alpha_T \Delta T$$

We can define bimorph actuation sensitivity  $\gamma$

$$\gamma = \theta / \Delta T = \beta_r L_b \Delta\alpha_T$$