

Fastening (more complex shapes = better function)

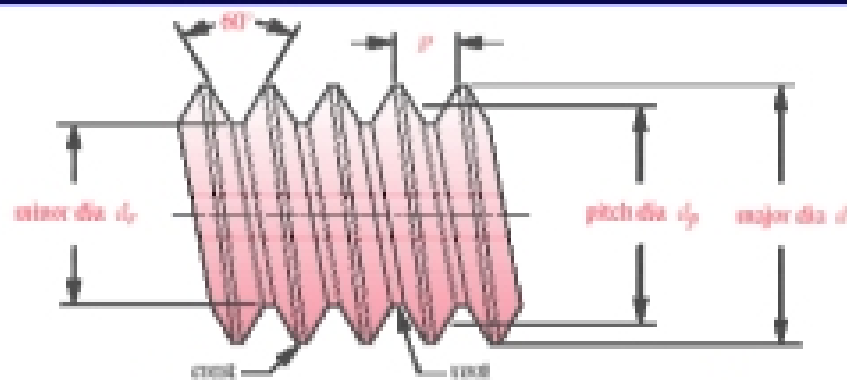
- ❖ Non-permanent
 - Bolted

- ❖ Permanent
 - Bolted
 - Welded
 - Bonded

Outline

- ❖ General Thread Nomenclature & Types
- ❖ Power Screws
- ❖ Stresses in Threads
- ❖ Preloading Fasteners/Joints
- ❖ Fasteners in Shear

Threads



p	pitch	In./thread
d	diameter (major)	In.
d_p	pitch diameter	In.
d_r	minor diameter	In.
L	Lead	In.

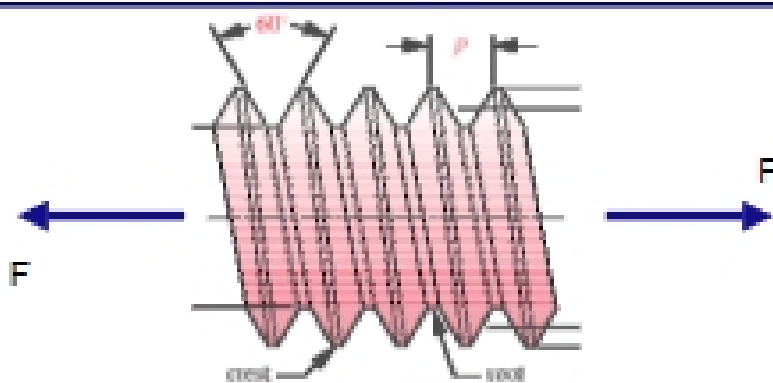
Screw Classifications

	Unified National Standard	ISO (Metric)
Thread Pitch	UNC -coarse UNF -fine UNEF -extra fine	coarse fine
Tolerance	Class 1 Class 2 Class 3	several levels

$d=0.25"$	fine	Class 2	$d=12\text{mm}$
20 threads/in.	1/4-20 UNF -2A	external threads	metric M12 x 1.75
			$p=1.75\text{ mm/thread}$

see Tables 14-1 and 14-2 for standard sizes

Tensile Stress



$$\sigma_t = \frac{F}{A_t}$$

$$A_t = \frac{\pi}{4} \left(\frac{d_p}{2} + \frac{d_r}{2} \right)^2$$

A_t also in Tables 14-1 and 14-2

Outline

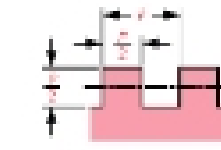
- ❖ General Thread Nomenclature & Types
- ❖ Power Screws
 - Threads
 - Loads
 - Self-locking
 - Efficiency
- ❖ Stresses in Threads
- ❖ Preloading Fasteners/Joints
- ❖ Fasteners in Shear

Power Screw Applications

Where have you seen power screws?

- ❖ jacks for cars
- ❖ C-clamps
- ❖ vises
- ❖ Instron material testing machines
- ❖ machine tools (for positioning of table)

Power Screw Types



- ❖ **Square**
 - > strongest
 - > no radial load
 - > hard to manufacture



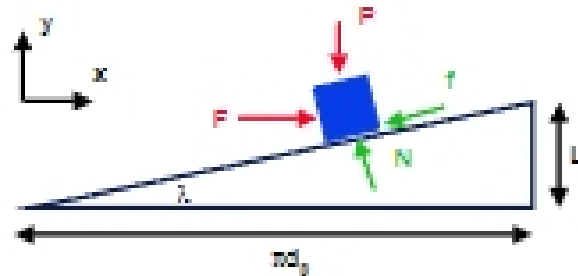
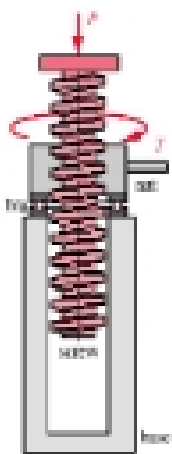
- ❖ **Acme**
 - > 29° included angle
 - > easier to manufacture
 - > common choice for loading in both directions



- ❖ **Buttress**
 - > great strength
 - > only unidirectional loading

Load Analysis

What "simple machine" does a power screw utilize?



LIFTING

$$\tan \lambda = \frac{L}{\pi d_p}$$

$$T S_o = \frac{P d_p (\mu \pi d_p + L)}{2 (\pi d_p - \mu L)}$$

Friction Coefficients

$$\mu_{\text{oil lubricated}} = \mu_{\text{collar w/ bushing}} = 0.15 \pm 0.05$$

$$\mu_{\text{collar w/ bearing}} = 0.015 \pm 0.005$$

Ball Screw

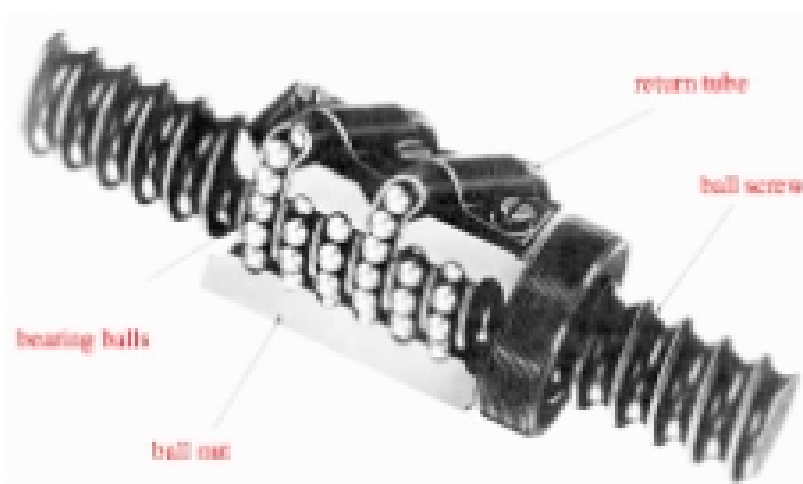


FIGURE 14-9

Outline

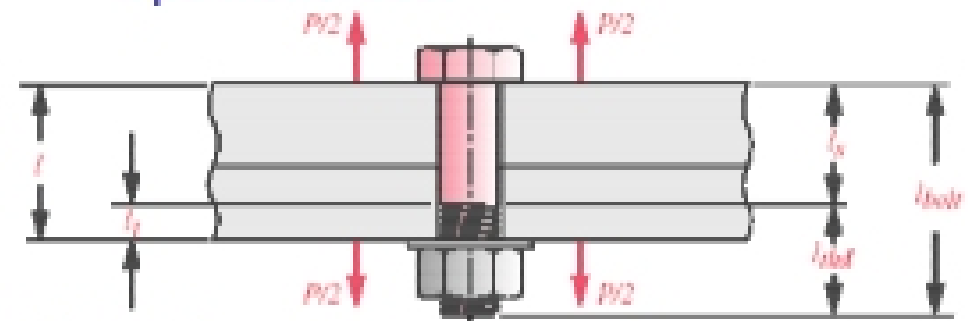
- ❖ General Thread Nomenclature & Types
- ❖ Power Screws
- ❖ **Stresses in Threads**
 - > **Body Stresses**
 - Axial
 - Torsion
 - > **Thread Stresses**
 - Bearing
 - Bending
 - > **Buckling**
- ❖ Preloading Fasteners/Joints
- ❖ Fasteners in Shear

Outline

- ❖ General Thread Nomenclature & Types
- ❖ Power Screws
- ❖ Stresses in Threads
- ❖ Preloading Fasteners/Joints
 - Proof Strength
 - Spring Behavior
 - Loading & Deflection
 - Separation of Joints
- ❖ Fasteners in Shear

Preloading & Proof Strength

- ❖ $S_p \rightarrow$ stress at which bolt begins to take a permanent set

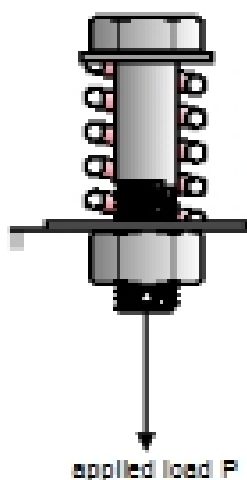


Preloading

- static loading: preload at roughly 90% of S_p
- dynamic loading: preload at roughly 75% of S_p

Spring Behavior

BOTH material being clamped and bolt behave as springs (up to yield/permanent set stresses)



$$k = \frac{AE}{l}$$

for the bolt, threaded vs unthreaded have different spring constants:

$$\frac{1}{k_b} = \frac{l_t}{A_t E_b} + \frac{l_s}{\frac{\pi d^2}{4} E_b}$$

Strategy Reviewed

See Example 14-2, p. 906

Given: joint dimensions

Find: bolt

set preload equal to 90% S_p
find l_t so that you can find k_b
find k_m

calculate C, then F_b , F_m then F_b , F_m
find stress in bolt and separation load

Such that: factors of safety > 1

Dynamic Loading of Fasteners

- ❖ Bolt only absorbs small % of P
- ❖ Stresses
 - Bolt is in tension
 - Material is in compression
- ❖ Fatigue is a tensile failure phenomenon
- ❖ \therefore Preloading helps tremendously in fatigue

Outline

- ❖ General Thread Nomenclature & Types
- ❖ Power Screws
- ❖ Stresses in Threads
- ❖ Preloading Fasteners/Joints
- ❖ **Fasteners in Shear**
 - What is Shear?
 - Straight Direct Shear